

Early-Detection of EVA Encapsulant Degradation in PV Modules Based on Vibration Frequency Analysis

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Abstract

In engineering applications, the frequency analysis represents a first and practical step to collect relevant parameters for structural and mechanical diagnostics. Any possible material / component degradation and deterioration can be prematurely detected by frequency modifications that exceed a certain alert value. In this paper, the attention is given to the dynamic mechanical analysis of commercial photovoltaic (PV) modules, in which the solar cells are typically encapsulated in thin viscoelastic interlayers made of Ethylene-Vinyl Acetate (EVA), which are primarily responsible for the load-bearing capacity of the sandwich PV system. As a major effect of ageing, ambient conditions, nonuniform / cyclic thermal gradients, humidity and even extreme mechanical / thermal loads, the rigidity of these films can largely modify and decrease, thus possibly affecting the mechanical capacity of the PV module, and even exposing the solar cells to fault. Knowledge of the effective bonding level is an important step for diagnostic purposes. In this regard, the present study is based on a preliminary but extensive parametric Finite Element (FE) numerical investigation of full-scale commercial PV modules of typical use in buildings. The attention is given – for PV module arrangements of technical interest – to the effect of EVA stiffness in terms of vibration modes and especially frequency sensitivity. As shown, when compared to newly installed PV modules, any kind of stiffness decrease is associated to major frequency modifications for the composite system, and in the worst configuration, such a frequency scatter can decrease down to -40% the original condition. Such a marked stiffness decrease would be implicitly associated to a weak mechanical performance of the sandwich section, with major stress peaks and deflections in the PV system, even under ordinary loads. The presented results, in this sense, suggest that major consequences can be prevented and minimized by monitoring the vibration frequency of PV modules.

Keywords

Photovoltaic (PV) modules, glass, Ethylene-Vinyl Acetate (EVA), mechanical properties, materials, Finite Element (FE) numerical modelling, vibration frequency

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1. Introduction

Photovoltaic (PV) modules are designed to ensure more than 25 years of functionality under variable and even unfavourable operational conditions, including the effect of temperature / humidity changes, wind, rain, hail and snow, etc. While their electrical functionality is of primary importance, the mechanical performance also represents a critical aspect, which is implicitly correlated to the durability and efficiency of PV modules. According to literature, in this regard, there are several monitoring and diagnostic strategies to monitor possible faults and defects, such as shading, dust or sand accumulation on the PV surfaces, short circuits, bypass diodes failures, disconnections of PV modules, shunting of PV modules, and others (Mellit et al. 2018, Li et al 2019, Mellit et al. 2023). Hot spot faults represent one of typical localized damage configurations that should be promptly detected (Tsanakas and Botsaris 2013, Zhang et al. 2017, Jiang et al. 2020, Li et al 2022).

Structurally speaking, a special care should be required for the prompt detection of many different progressive degradation / failure mechanisms that are associated to possible power-losses and major effects on the functionality and efficiency of PV modules, such as possible micro and macro cracks in cells (Sander et al. 2013), breakage of interconnects (Dietrich et al. 2013), detachment of the frame members at the edges, and especially delamination of sandwich components in the PV system (Dadaniya and Varma Datla 2020, Dobra et al. 2022, Sangpongsanont et al. 2023, Meena et al. 2024, Bedon et al. 2024).

2. Reference PV module and methodology

2.1. Geometry

As in Fig. 1, the majority of commercial PV modules are usually characterized by a relatively thin crosssection and a typically high slenderness and bending flexibility to out-of-plane mechanical loads, due to their size-to-thickness ratio and also to the typical arrangement of restraints and fixings. As a sandwich cross-section, the mechanical capacity is usually assigned to thin glass covers with a typical thickness in the range of $h_1 \approx 3-5$ mm and (for single-glass compositions) back Tedlar[®] / plastic layers with $h_2 \approx 1-2$ mm, which are mechanically bonded by Ethylene-Vinyl Acetate (EVA) interlayers with $h_{int} \approx$ 1 mm in total.



Fig. 1. Reference commercial PV module (example): (a) axonometry and (b) schematic cross-section.

2.2. Strategy

Following the existing literature evidences, a Finite Element (FE) numerical study is thus presented in this paper to assess the convenience of non-destructive dynamic techniques for the mechanical characterization and for the early damage / deterioration detection of EVA bonding interlayers for PV modules (Fig. 2). The methodology takes advantage of classical structural health monitoring (SHM) procedures for civil structures and machineries (Pander et al. 2011, Limongelli et al. 2021, Bedon et al. 2023, Dimarogonas 1996), where the on-site experimental analysis in the frequency domain is usually carried out to detect and characterize the vibration modes of the system object of study, and thus to assess the effect of single components and possible influencing parameters. The approach follows earlier applications to laminated glass systems and composite elements, where in-field and/or historic structures have been efficiently characterized under various loading and boundary conditions, including walkways in unfavourable environment and operational configurations (Bedon 2019a), facades (Bedon et al. 2022), fractured pedestrian systems (Bedon et al. 2021) and beams with delamination (Bedon 2019b).



Fig. 2. Scheme of present methodology and investigation.

The present study, differing from earlier investigations, consists in the analysis of composite systems characterized by higher flexibility (due to typical geometrical and mechanical features), severe exposure to environment, and thus higher sensitivity to possible material degradation. A careful calibration of a FE numerical model representative of the reference PV module is performed, in which the basic constituent members are efficiently described in geometrical and mechanical terms, and the EVA material is described as equivalent linear elastic. Parametric numerical analyses are hence carried out in Section 5 to address the sensitivity of dynamic features to basic material modifications for the bonding encapsulant, which could take place due to progressive ageing in the long-term period, exposure to unfavourable ambient conditions, repeated thermal gradients, and even possible localized / non-uniform delamination or damage due to extreme mechanical loads.

3. Frequency analysis

The current approach assumes that the frequency analysis of PV modules – experimentally derived based on the Fourier Transform function analysis of measured acceleration records, as in Fig. 2 – can offer important feedback and can be used to calibrate and validate a refined FE model representative (in average) of the tested PV samples, in which the geometry and the mechanical composition of a selection of commercial full-scale modules is properly taken into account. The numerical analysis of the reference FE model is then further extended to a parametric and sensitivity study, to assess the effect of further influencing parameters of technical interest for long-term and severe ambient conditions.

Under unfavourable environments, the vibration frequency of the PV system in Fig. 3 (a) is in fact affected by many factors, that are namely associated to the sandwich section features, to the contribution and edge restrain of the metal frame (Fig. 3 (c)), to the effect of brackets (Fig. 3 (d)), etc. At the level of sandwich in Fig. 3 (b), the protective glass cover is expected to behave elastically without significant modifications in the elastic modulus of the basic constituent material (Udi et al. 2023), and the same concept applies to the aluminium components of frame members and restraints. Besides, the interposed EVA films are sensitive to temperature and humidity, which could both affect the PV modules in the long-term period (Dietrich et al. 2010, Hána et al. 2019). As far as the bonding interface of EVA films and glass/plastic covers is possibly affected by external agents, and/or the interlayer suffers for stiffness modification due to operational conditions, the PV module acts as a composite panel with weak / flexible shear connection, and this could result in compromised load-bearing mechanisms under design mechanical loads (Naumenko and Eremeyev 2014, Gong et al. 2021). On the other side, it is experimentally proved that high vibration frequencies for EVA films - like for viscoelastic interlayers in general - typically have a stiffening effect, and this can be positively quantified in a possible moderate increase in shear bond efficiency (Knight et al. 2022, Hána et al. 2019). However, it is also experimentally observed that repeated mechanical loads (even at high strain rate) induce a marked reduction in the material stiffness, with deterioration of shear bond rigidity (Schmidt et al. 2017). In such a variable and uncertain scenario, both the metal frame at the edges (Fig. 3 (c)) and the fixing brackets (Fig. 3 (d)) – that can be efficiently characterized by numerical analysis – further contribute to modify the vibration response of the PV system, compared to ideal simplified restraints.



Fig. 3. Influencing parameters in the frequency analysis of a PV module: (a) reference system, (b) sandwich panel, (c) frame, (d) brackets.

To assess the severity of all these possible effects, the present study assumes that – for the sandwich section – the stiffness of EVA films and its viscoelastic modification are for the major part responsible of the vibration frequency and rigidity of a given PV module in out-of-plane bending deformations. The "effective" composite flexural stiffness of the sandwich is in fact (Li et al. 2020, Gong et al. 2021, CNR-DT 210/2013):

$$D_{eff} \approx f(glass, EVA, Tedlar)$$
 (1)

Rationally, it is expected that:

$$D_{abs} \leq D_{eff} \leq D_{full}$$

where D_{abs} and D_{full} denote the bending stiffness values of the sandwich section under weak / null shear bonding offered by EVA foils ("abs", layered limit) or with ideally rigid connection in shear ("full", monolithic limit) respectively. From an analytical point of view, as also in accordance with Li et al. (2020) and other studies, the basic assumption for the analytical mechanical analysis of a sandwich section like in Fig. 3 (b) is that:

- the front glass cover (h_1 thickness) is described as linear elastic layer (with E_1 , v_1 , ρ_1 the material properties in terms of modulus of elasticity, Poisson' ratio, density);
- the backboard Tedlar layer (h_2 thickness) can be also described as linear elastic (E_2 , v_2 , ρ_2);
- the EVA interlayer, with total thickness h_{int}, is described as an equivalent linear elastic material with secant stiffness E_{int} which is in general very soft compared to E₁ and E₂. For more realistic estimates, however, this value should be calibrated to represent based on the typical viscoelastic material behaviour a specific loading and boundary condition of practical interest (i.e., time loading, temperature, ageing (CNR-DT 210/2013));
- the encapsulated solar cells are disregarded in mechanical terms, and assumed to offer null contribution to the composite PV section in bending.

(2)

Under such a kind of assumption, it is conservatively expected that the bending stiffness of the sandwich section is mostly governed by the front and back covers only, where (i = 1, 2):

$$D_i = \frac{E_i h_i^3}{12(1-\nu_i^2)}$$
(3)

is the stiffness of external layers and the limit values are:

$$D_{abs} = \sum D_i \tag{4}$$

$$D_{full} = D_{abs} + \frac{12D_1D_2}{D_1h_2^2 + D_2h_1^2}H^2$$
(5)

with:

$$H = h_{int} + \frac{h_i}{2} \tag{6}$$

The corresponding deflection under uniform pressure and simple boundary conditions can be thus estimated analytically, see for example Naumenko and Eremeyev (2014).

4. Numerical modelling

The reference FE model was assembled in ABAQUS (Simulia). To optimize the computational cost of simulations, the FE model consisted of a combination of full 3D solid brick elements (for the metal frame and brackets) and 2D shell elements for the PV components schematized in Fig. 1 (i.e., glass cover, EVA, Tedlar layer, solar cells). To account for possible complex vibration modes, the full geometry of PV modules was numerically described. The adopted mesh pattern was chosen to ensure accuracy in frequency estimates, and thus resulted in \approx 28,000 elements and \approx 42,000 DOFs, see Fig. 4.

Materials were described in the form of linear elastic constitutive laws, for the purpose of linear modal analyses and frequency predictions. For glass and aluminium, the nominal mechanical properties in Table 1 were taken into account (EN 572-2, EN 485-2). The Tedlar plastic layer was mechanically characterized based on product datasheets, see Table 1 and Tedlar Technical Bulletin (2020). Finally, for the EVA interlayer, the tentative mechanical characterization was preliminary based on experimental and numerical studies reported in Li et al. (2020) and Knight et al. (2022). To note that in the present analysis, the solar cells were described as an equivalent linear elastic material (Li et al. 2020).



Fig. 4. Numerical model (ABAQUS): (a) assembly concept ($1/4^{th}$ geometry with symmetry boundaries) and (b) example of axonometric view for the PV module with mounting system (α = 40°).

		Glass	Aluminium	Tedlar	EVA	Solar cells
E	[GPa]	70	69	25	0.005	170
n	-	0.23	0.3	0.3	0.49	0.25
r	[kg/m3]	2490	2700	1000	950	2330

Table 1. Input mechanical properties for the frequency analysis (ABAQUS).

5. Preliminary results

The parametric analysis was carried out to assess the effect of a progressive modification of the interlayer stiffness. For the purpose of present study, the elastic modulus E_{int} in Table 1 was progressively modified in the range from 0.1 MPa up to 70 GPa, which correspond respectively to the "abs" lower limit and to the ideal "full" configuration in which the interlayer has the same rigidity of the glass front cover.

In terms of vibration modes and shapes of the PV module, no marked modifications were noted. Besides, a major effect of interlayer stiffness was numerically quantified in the modification of vibration frequencies for the examined PV module. In this regard, the typical results can be seen in Figs. 5 (a) to (d) in terms of frequency trend as a function of E_{int} , for the four lower vibration modes, which are characterized by one-to-three half sinewaves for the PV module. As far as E_{int} ideally increases, the corresponding vibration frequency progressively increases for all the detected vibration modes.

Most importantly, Fig. 6 further emphasizes the interlayer effect on the vibration frequency, as a function of the "abs" or "reference" (Table 1) shear bonding level offered by the interlayer.

Compared to a weak bond in Fig. 6 (a), it can be seen that the minimum contribution offered by the interlayer can manifest in a major stiffening contribution for the sandwich section. The calculated percentage scatter of frequency increases tends to increase with the increase of mode order, that is the number of sinewaves and the bending deformation of the system.

Besides, especially for health monitoring purposes, the percentage frequency variation can be further interestingly measured as a function of the "initial" configuration of the PV module, that is the system with efficient bonding connection at the time of its installation. For the present study, the numerical results in Fig. 6 (b) are calculated in percentage scatter compared to the E_{int} properties reported in Table 1. It can be thus noted that even a major increase in the stiffness of interlayer (i.e., as a function, for example, of high dynamic or impulsive loads), there is no marked modification in the corresponding vibration frequency. Only for very rigid (but not realistic) bonds, it can be expected a frequency increase up to 13-15% for the three lower vibration modes of the system. It is worth to note in Fig. 6 (b) that event relatively stiff bonds (i.e., up to \approx 1000 MPa in modulus) would result in a minimum increase of stiffness and frequency (less than 3%). This suggests that the sandwich system under high strain rate effects (i.e., for the PV module under impact loading) cannot take advantage of possible stiffening effects in the interlayer.



Fig. 5. Vibration frequency of the PV module as a function of E_{int} (α = 40°): (a) modes 1, (b) 2, (c) 3, and (d) 4.



Fig. 6. Percentage variation of vibration frequency of the PV module as a function of E_{int} (α = 40°) for modes 1, 2, 3 and 4, based on: (a) "abs" limit and (b) "Ref" value.

It is indeed of major interest the effect of bond rigidity when degrading phenomena, as well as the progressive ageing of interlayer stiffness or even possible delamination of interfaces, can take place in the long-term period. Fig. 6 (b) highlights in fact a frequency decrease down to -45% when the shear bonding efficiency of the connection vanishes. Also, the higher is the mode order for the examined system, and the higher is the percentage variation of vibration frequency.

In other words, the present analysis confirms that the interlayers have – as in typical glass and composite laminates applications – a key role for the determination of the load-bearing capacity and mechanical performance of typical thin sandwich sections in use for commercial PV modules. In the long-term period, or even in unfavourable environments that could affect the interlayer efficiency, it can be useful to track and detect possible modifications in the mechanical properties of basic components, and thus prevent possible weaknesses or loss or functionalities for PV modules.

6. Conclusions

Photovoltaic (PV) modules, that are designed to ensure about 20-25 years of functionality, are typically exposed to variable environment scenarios and even unfavourable operational conditions, including the effect of temperature changes, wind, rain, hail and snow, etc. Their mechanical efficiency and performance, in this regard, represents a critical aspect to support an optimal durability and electrical efficiency of PV modules. Besides, typical sandwich sections are composed of relatively thin top/bottom covers and an interposed film to encapsulate the solar cells. From a mechanical point of view, the interlayers have a key role in ensuring the shear bond between the constituent components, and thus maximize the load-bearing performance capacity of the system. Among others, the bonding interlayers can suffer for stiffness modification which has direct effects on the composite action and response of the sandwich section.

The present investigation was focused on the frequency assessment to monitor and alert possible major modifications in the encapsulant layers, to prevent major stress peaks and possible severe damage conditions in the sandwich section. The numerical study proved that the vibration frequency can be largely sensitive to possible stiffness modifications in the interlayers, with frequency variations down to -40%, and thus should be usefully checked to reveal the progressive mechanical deterioration or even possible defects in the sandwich components. To this aim, the present numerical analysis will be further exploited by including in the investigation a number of different PV module configurations, as well as experimental observations from in-field measurements on aged PV samples.

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