

# Stress Distribution along the Structural Sealant Joint Length of a Cylindrically Curved Glazing Panel

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## Abstract

It has been identified that current standardised method for structural sealant joint dimensioning is applicable to flat rectangular panels only and no provisions are made for panels with curved surfaces. The purpose of this study is to investigate the stress distribution along the sealant joint of a cylindrically curved glass panel subjected to wind pressure and to establish if the panel curvature influences the stress distribution along the joint length. Using numerical method, several curved units were analysed and the results have shown that the out of plane wind action generate compression and shear forces within the joint, both occurring at the same time. This means that in the case of a structurally bonded curved panel, additionally to the shear caused by differential thermal expansion of elements or glass self-wight which is covered by ETAG 002 and EN 13022 the designer must also consider the shear stress induced due to the panel curvature. The magnitude and location of this additional shear stress and also of the tension/compression stress can be identified using Finite Element Analysis.

## **Keywords**

Structural Sealant Glazing, Cylindrically Curved Structural Sealant Glazing, ETAG 002, EN 13022, FEM.

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# 1. Introduction

The use of Structural Sealant Glazing system (SSG) in large glass facades is a well-established construction method in current practices (Alcaine et al. 2020). The methodology uses the glass bonding technique to redistribute the applied loads to the façade structure via perimetral structural sealant applied in between the glass infill panel and metallic subframe (ETAG 002 2012). The method originates in 1965 and has been used in projects around the world that have performed well for more than 40 years (Clift et al. 2015).

The application of structural sealant can be realized on site or in the factory. Site bonding typically is not allowed by the specifiers as the procedure requires to be undertaken in a controlled environment, condition which cannot be provided onsite. The preferable option is the factory bonding where an adequate quality control procedure can provide a performant construction element (Klosowski and Wolf 2015).

# 2. Background

In Europe the mechanical performance of the SSG joint is determined by following the standardised analytical approach which is regulated by ETAG 002 (Wurm J. 2007), EN 13022 and EN 16759. The joint size is a variable depending on glass size, wind load, sealant strength (Van Lancker B. 2020) and is calculated assuming homogeneous stress distribution across the sealant bite (Descamps and Hayez 2018) rigid supports, uniform load distribution, small deformations and linear material behaviour (Clift et al. 2015; Drass and Kraus 2020).

Despite the continuous improvement of the sealants since the 1970's and the current availability of high-performance structural adhesives the calculation method did not change since (Van Lancker B. 2020). This is currently challenged by the industry due to conservative assumptions used in stress analysis and the high global safety factor  $\gamma_{tot} = 6$  specified by ETAG 002 (Drass and Kraus 2020). Recent studies propose the dimensioning of SSG assuming a nonlinear distribution of the stresses across the sealant bite (Clift et al. 2015) and by modelling the joint using hyperplastic material (Alcaine et al. 2020; Clift et al. 2015) or by using a spring model to calculate the deformation of structural joints (Descamps et al. 2020). On the other end (Drass and Kraus 2020) are proposing a new Eurocode-compliant design concept for the sealant joint and determine a lower global partial safety factor  $\gamma_{tot}^* = 2.72$  based on experimental data and for a specific sealant type.

In the case of cylindrically curved structural sealant glazing none of the design rules from the current standards or guideline are applicable as the dimensioning concept according to ETAG 002 is valid only for rectangular, flat glazing (Heinze et al. 2016). In case the design conditions are out of the standard scopes the analysis of stresses and strains within the sealant is recommended to be assessed with the help of Finite Element Method (Memari et al. 2021), (Clift et al. 2015). However, there is no standardised methodology to run FEA for evaluation of SSG but notwithstanding that the method is widely used within the industry as it allows to predict the local stress distribution within the sealant volume (Descamps et al. 2017).

The ETAG 002 and EN 13022 limits the SSG construction to four main types depending on the glass infill and the arrangement of mechanical restraint devices. In all four instances the glass is bonded to the support frame along the perimeter and the out of plane loadings are always transferred to the support frame via bonded connection (ETAG 002 2012).



Fig. 1: ETAG system types (Wurm J. 2007).

For types I and II only short-term out of plane loads are transferred by the adhesive where for the permanent in plane loading transfer the use of mechanical self-weight supports is mandatory. The type III and IV are approved for single glazing only as both permanent and variable loadings are allowed to be taken by the structural bond only. In case redundancy is required, the system type I and III have additional mechanical out of plane retaining devices specified which will reduce the danger in case the adhesive fails.

The use of type II SSG in building facades has grown significantly in recent years as it offers a continuous architectural smooth glass surface across the building exterior while providing safety, enhanced thermal, acoustic and blast performance to the building (Alcaine et al. 2020). The use of this SSG type is also observed in the case of facades employing curved glass panels although the design of these is not mentioned and no provisions are made for this panel shape by any standard or guideline.



Fig. 2: IPC House, Dublin – curved SSG (GLF Facades).

Fig. 3: The Malthouse, Dublin - curved SSG (GLF Facades).

The behaviour of the available adhesives is not fully comprehended and the current standards and guideline limit the design options to a series of rules proven to be adequate over the time (Clift et al. 2015). According to (ETAG 002 2012) the sealant joint should be made of structural silicone only, should be rectangular in cross section and applied continuously along the infill perimeter with the condition that the adhesion is made only on two parallel surfaces.

In a type II flat SSG the sealant bite  $h_c$  is a variable of the loads acting perpendicular to the glass surface which in turn generate tensile/compression stresses in the structural seal. The maximum stress magnitude is considered to develop at the centre of the longest side of the pane and can be calculated as follows:

$$h_c = \frac{aW}{2\sigma_{Ed}} \tag{1}$$

where: a - is the short side dimension of the glass pane, W - is the relevant combined actions of the wind, snow or self-weight and  $\sigma_{Ed}$  - is the allowable tensile design strength of the sealant.

In the similar manner the sealant thickness *b* is estimated by calculating the shear stress generated along the joint due to differential thermal expansion in between the metallic support frame and glass infill panel. The maximum stress magnitude is considered to develop in the corners of the pane and can be calculated as:

$$b = \frac{G\Delta}{\tau_{Ed}} \tag{2}$$

where: G - is the sealant shear modulus,  $\Delta$  - is the maximum thermal movement as a combination of elongation along both glass directions and  $\tau_{Ed}$  - is the allowable shear design strength of the sealant under dynamic loading.



Fig. 4: Structural sealant joint parameters - Type II

#### 3. Numerical analysis

Using linear analysis, a series of type II cylindrically curved structural sealant glazing units are investigated under the action of out of plane wind pressure to analyse the influence of panel curvature on the stress distribution along the joint length. The panels are classified in three categories, "tight", "shallow" and "almost flat" units depending on the achievable bending angle. Each of the size is referred to the manufacturing capabilities of the glass processors listed in (Timm C and Chase J 2014). The variables presented in table 1 are for 6 mm thick monolithic hot bent tempered glass considering the glass is curved in convex shape, i.e. the curvature radius and performance coating faces the inner side of the building.

	Classification depending	Minimum/maximum	Maximum arc length "A"	
Specimen	on bending radius	bending radius "R"	depending on the minimum/maximum	
	[mm]	[mm]	bending radius [mm]	
1.1	Units with "tight radius" (minimum achievable radius)	650	1200 at α=105°	
1.2		700	1700 at α=140°	
1.3		1000	1900 at α=110°	
2.1	Units with "shallow radius" (mid-range radiuses)	2000	2700 at α=84°	
2.2		3000	3000 at α=57°	
2.3		4000	3200 at α=46°	
3.1	"Almost flat units" (maximum achievable radius)	10000	3200 at α=18°	
3.2		20000	3200 at α=9.5°	
3.3		25000	2400 at α=5.6°	



Fig. 5: Geometrical variables of cylindrically curved panel (ISO 11485-1:2011).

The specimens are modelled by means of geometrically linear finite element analysis using software package RFEM v.5.27.01 and 2D rigid plate elements. The units are continuously pinned along the perimeter with fixed restraints against any translational displacements along ux, uy, uz and free to rotate around  $\varphi x$ ,  $\varphi y$ ,  $\varphi z$ . The pressure magnitude considered equals to +/-2.4 kPa which refers to the value used for testing of facade performance mock-ups for safety (CWCT 2006) and which is applied perpendicular to the true area of the unit, upwind.



Fig. 6: SSG joint, type II (left). SSG – static model (centre). SSG – Finite Element Model (right).

Under the action of out of plane wind loading the line supports exert a linear reaction which is further called "line force" and is used to illustrate the stress distribution pattern and intensity along the structural sealant joint. Using these out of plane line force magnitudes the maximum normal tension/compression stress can be calculated as follows:

$$\sigma_{max} = \frac{R_{z,max}}{h_c} \tag{3}$$

where:  $\sigma_{max}$  - is the maximum tensile stress calculated using maximum out of plane line force magnitude,  $R_{z,max}$  - is the maximum line force magnitude acting perpendicular to the glass plane,  $h_c$  - is the designed sealant bite.

In the case of other ETAG types the maximum shear stress along the joint can be calculated using inplane line forces:

$$\tau_{max} = \frac{R_{x,y,max}}{b} \tag{4}$$

where:  $\tau_{max}$  - is the maximum shear stress calculated using maximum in-plane line force magnitude,  $R_{x,y,max}$  - is the maximum line force magnitude acting in glass plane, b – is the designed sealant thickness.

To validate the Finite Element model, a flat unit is simulated and the results are checked using analytical formulas (1) to (4). The unit measure 4200 mm in height and 3200 mm in width which equals to the arc length of the specimen 3.2 and the goal is to achieve the distribution pattern described in ETAG 002 or EN 13002.



Fig. 7: Flat specimen (ETAG 002, type II system) - Line forces pz (out of plane).

As it can be seen the line force distribution pattern and the maximum magnitude is observed along the longest edge which is like the pattern described in ETAG 002 and EN 13022. Following the same conditions, the specimens presented in table 1 were modelled and the line force magnitude results are summarised in table 2. For illustrative and proof purposes, the line forces for three out of nine specimens/models are presented further.



Fig. 8: Specimen 1.1. Out-of-plane line forces pz (left). In-plane line forces py (centre). In-plane line forces px (right).



Fig. 9: Specimen 2.1. Out-of-plane line forces pz (left). In-plane line forces py (centre). In-plane line forces px (right).



Fig. 10: Specimen 3.2. Out-of-plane line forces pz (left). In-plane line forces py (centre). In-plane line forces px (right).

# 4. Findings

In the case of curved panels, it can be observed that the distribution of line forces along the perimeter are different than in the case of flat panel. The ETAG type II flat panel subjected to wind pressure generates only line forces along the local z axis " $p_z$ " and no line forces along the local y or x axis are recorded. However, in the case of curved panels the out of plane action of the wind pressure generates not only " $p_z$ " response but also in plane line forces along y axis " $p_y$ " and along x axis " $p_x$ " indicating that beside the short-term pure tension-compression state in the joint there will be shear forces induced at the same time. The maximum shear stress is dominant in magnitude in the case of "tight" and "shallow" units and occur always at the mid height of the straight edge (sample 2.1) whereas for the "almost flat units" the tension/compression forces prevail in magnitude (sample 3.2).

	Line force "p <sub>z</sub> " (N/mm)		Line force "py" (N/mm)		Line force "p <sub>x</sub> " (N/mm)	
_	tension/compression forces		shear forces		Shear forces	
	Curved edge	Straight edge	Curved edge	Straight edge	Curved edge	Straight edge
	(at vertex)	(at mid height)	(at vertex)	(at midspan)	(at 1/3 of length)	(at 1/8 of height)
Flat	3.59	3.83	0	0	0	0
1.1	0.66	0.51	0.22	1.46	0.28	0.1
1.2	0.79	0.40	0.41	2.25	0.51	0.17
1.3	0.97	0.47	0.41	2.64	0.48	0.18
2.1	1.68	1.06	0.65	4.33	0.69	0.33
2.2	2.22	2.02	0.57	3.98	0.52	0.31
2.3	2.10	2.25	0.42	3.98	0.34	0.22
3.1	3.35	3.54	0.25	1.79	0.19	0.15
3.2	3.63	3.82	0.14	1.01	0.11	0.08
3.3	2.62	3.08	0.06	0.41	0.04	0.03

Table 2: Line forces – Peak magnitude of line forces under the action of positive/negative wind pressure.

When comparing to the force distribution patterns described in ETAG 002 and EN 13022 the tension compression forces distribution along the joint are qualitatively similar in both flat and curved cases but quantitatively varies in the units with tight and shallow radiuses as the maximum stress is recorded at the centre of the curved sides which is the shortest in this case. Conversely, the shear forces distribution patterns disagree qualitatively as these develop at centres of the curved and straight edges in case of  $\tau_y$  and at 1/3 of the curved edge and the 1/8 of the straight edges in the case of  $\tau_x$ .

The curved units were also analysed separately under the influence of both positive and negative out of plane wind pressure this being applied perpendicular to the true area of the samples, upwind in the first case and downwind in the second and no differences in magnitude or distribution pattern were recorded.

#### 5. Conclusion

Current design rules for structural sealant glazing are related to flat rectangular panels only and no provisions are made for other panel shapes. Following FEM analysis, it was observed that the stress distribution pattern and magnitude along the cylindrically curved structural silicone joint length is different than the stress distribution pattern and magnitude along the flat structural silicone joint. The stress generated by the out of plane loading within the curved structural sealant glazing is multiaxial as the pressure generates not only normal out of plane stresses as in the case of the flat panels but also generates in-plane shear stresses which act simultaneously on the joint. This means that additionally to the shear caused by differential thermal expansion of elements or glass self-wight the designer must also consider the shear stress induced by the panel curvature under the action of wind pressure or any other out of plane loads. As the panel shape deviates from flat the magnitude and location of this additional shear stress and of the tension/compression stress can be identified using Finite Element Analysis.

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